

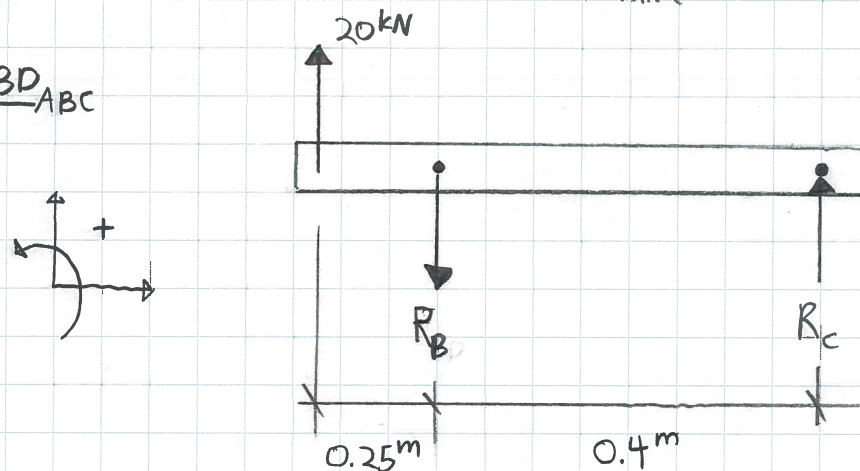
PROBLEM 1.27

For the assembly and loading of Prob. 1.7, determine (a) the average shearing stress in the pin at *B*, (b) the average bearing stress at *B* in member *BD*, (c) the average bearing stress at *B* in member *ABC*, knowing that this member has a 10 × 50-mm uniform rectangular cross section.

PROBLEM 1.7 Each of the four vertical links has an 8 × 36-mm uniform rectangular cross section and each of the four pins has a 16-mm diameter. Determine the maximum value of the average normal stress in the links connecting (a) points *B* and *D*, (b) points *C* and *E*.

1) Start with statics to determine the force in the pin at "B". Recognize links BD & CE are two-force members.

FBD_{ABC}

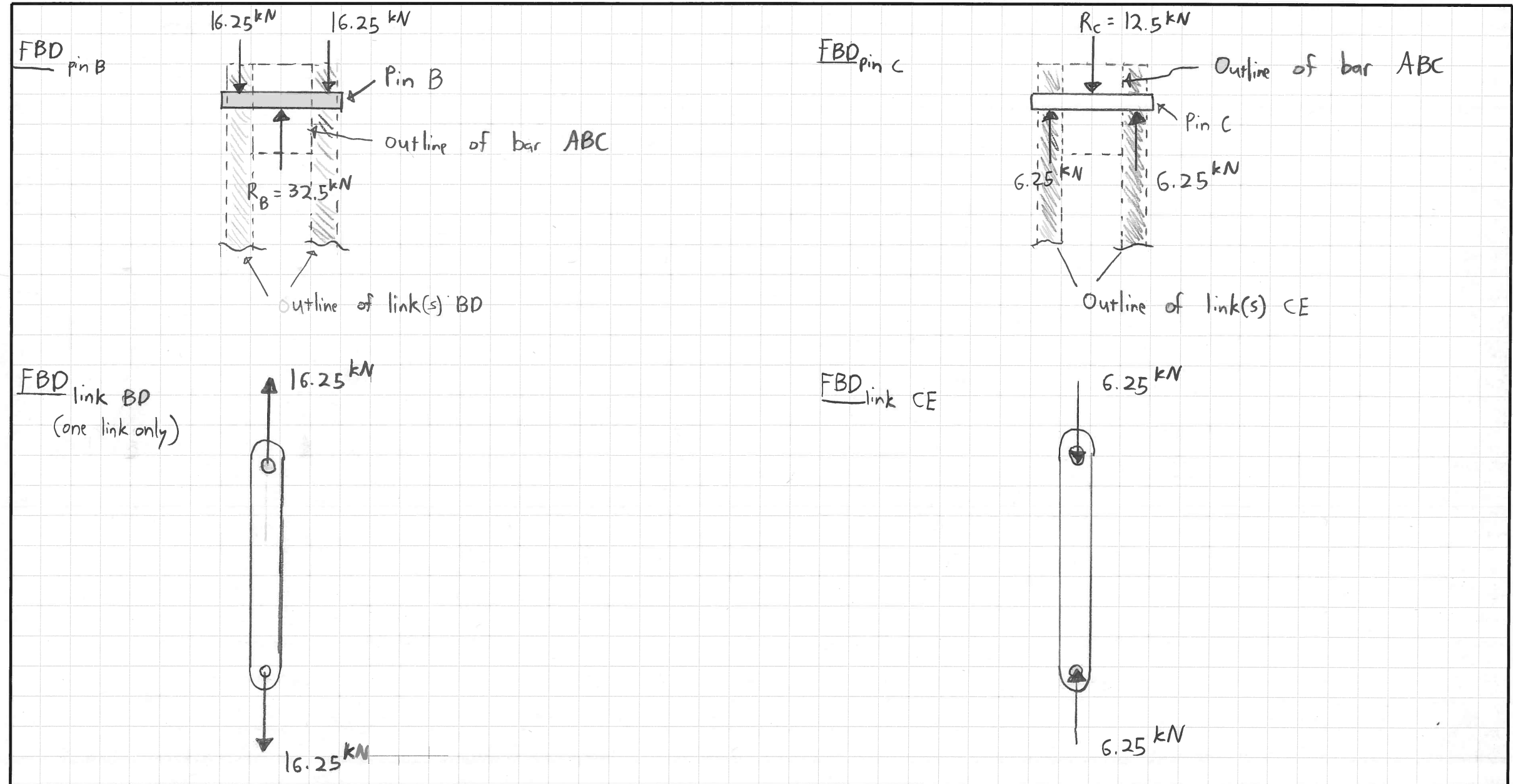


$$\sum \overset{+}{\curvearrowleft} M_C = 0; \quad R_B(0.4^m) - 20^{kN}(0.65^m) = 0$$

$$\underline{\underline{R_B = 32.5^{kN}}}$$

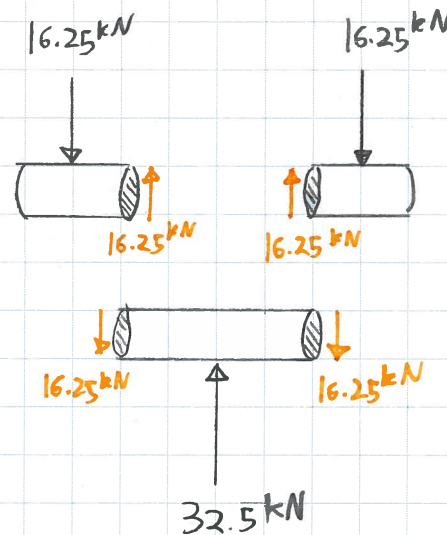
$$\sum \overset{+}{\curvearrowleft} M_B = 0; \quad -20^{kN}(0.25^m) + R_C(0.4^m) = 0$$

$$\underline{\underline{R_C = 12.5^{kN}}}$$



2) SHEAR STRESS IN PIN B

a) Failed Pin FBD



$$A_{\text{pin}} = \frac{\pi}{4} (16 \text{ mm})^2 = 201.06 \text{ mm}^2$$

a) So each shear plane must carry 16.25 kN

$$\tau = \frac{P}{A} = \frac{16.25 \times 10^3 \text{ N}}{201.06 \text{ mm}^2} = \underline{\underline{80.82 \text{ MPa}}}$$

b) Or, you can recognize the pin is in double shear (i.e. 2 shear planes to separate the free body) and use:

$$\tau = \frac{P}{2A} = \frac{32.5 \times 10^3 \text{ N}}{2 (201.06 \text{ mm}^2)} = \underline{\underline{80.82 \text{ MPa}}}$$

3) NORMAL STRESS IN LINK BD (AWAY FROM HOLES)

from free body, link BD must carry 16.25 kN

$$\sigma = \frac{P}{A} = \frac{16.25 \times 10^3 \text{ N}}{(8 \text{ mm})(36 \text{ mm})} = \underline{\underline{56.42 \text{ MPa}}} \quad (\text{Tension})$$

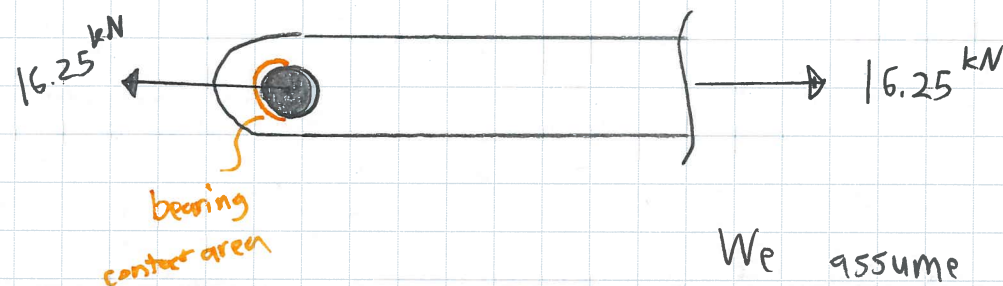
4) NORMAL STRESS IN LINK AT PIN B

$$\sigma = \frac{16.25 \times 10^3 \text{ N}}{(8 \text{ mm})(36 \text{ mm} - 16 \text{ mm})} = \underline{\underline{101.56 \text{ MPa}}} \quad (\text{Tension})$$

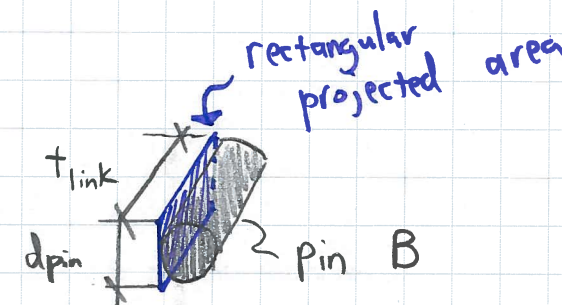
\nearrow t_{link} \nwarrow w_{link} \nwarrow d_{pin}

5) BEARING STRESS IN LINK AT PIN B

This is the stress between the pin & the link.



We assume \Rightarrow



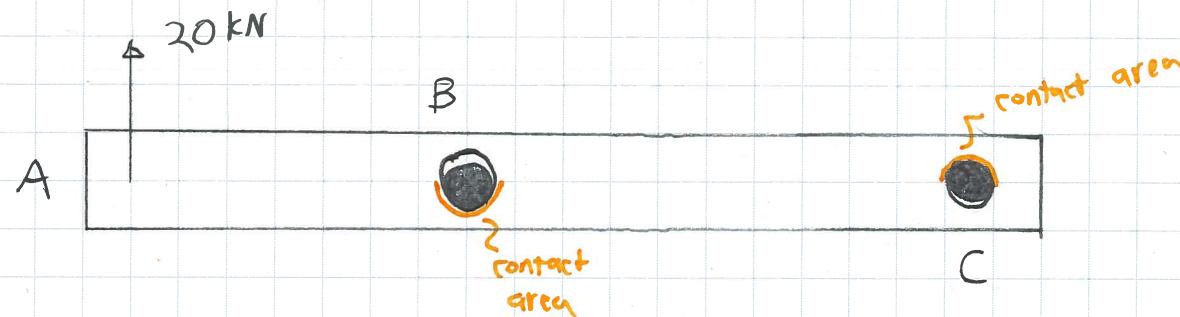
a) So,

$$\sigma_{\text{bearing}} = \frac{16.25 \times 10^3 \text{ N}}{\underset{\substack{\uparrow \\ d_{\text{pin}}}}{(16 \text{ mm})} \underset{\substack{\uparrow \\ t_{\text{link}}}}{(8 \text{ mm})}} = \underline{\underline{126.95 \text{ MPa}}} \quad (\text{Bearing is Compression})$$

b) Or, you can recognize that the 32.5 kN force on bar ABC is shared by two links at B. Thus,

$$\sigma_{\text{bearing}} = \frac{32.5 \times 10^3 \text{ N}}{(2 \text{ links})(16 \text{ mm})(8 \text{ mm})} = \underline{\underline{126.95 \text{ MPa}}}$$

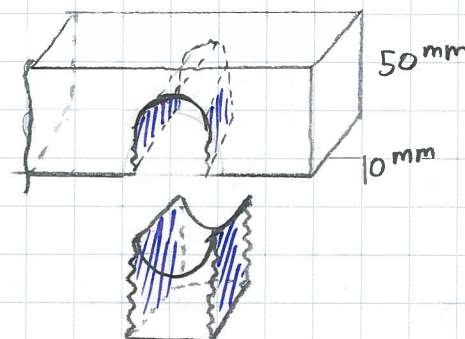
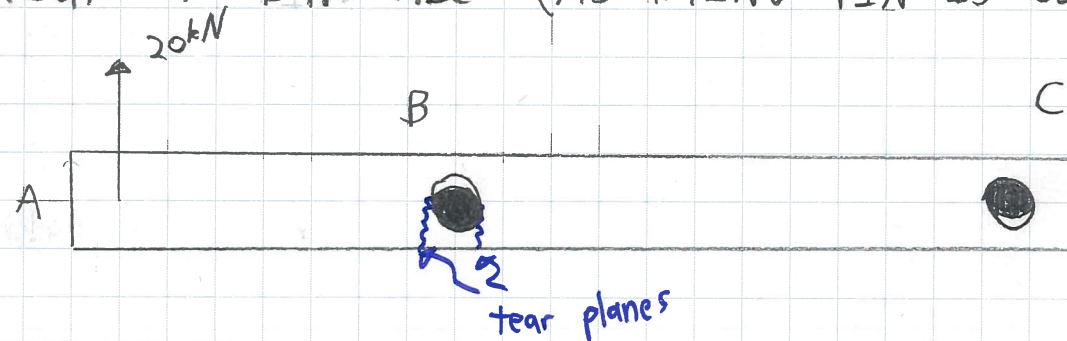
c) BEARING STRESS IN BAR ABC AT B



$$\sigma_{\text{bearing}} = \frac{32.5 \times 10^3 \text{ N}}{(10 \text{ mm})(16 \text{ mm})} = \underline{\underline{203.13 \text{ MPa}}}$$

\uparrow t_{bar} \uparrow d_{pin}

7) TEAR OUT OF BAR ABC (ASSUMING PIN IS CENTERED)



$$\sigma_{\text{tear}} = \frac{32.5 \times 10^3 \text{ N}}{(10 \text{ mm}) \left(\frac{50 \text{ mm}}{2} \right) (2 \text{ planes})} = \underline{\underline{65 \text{ MPa}}}$$

\uparrow t_{bar} \uparrow h_{hole}