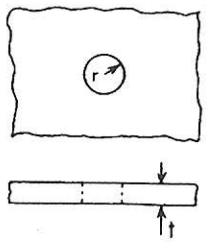
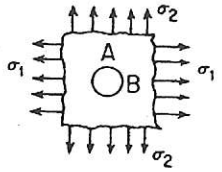
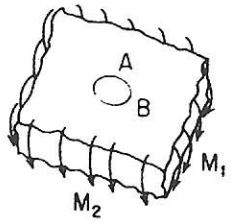
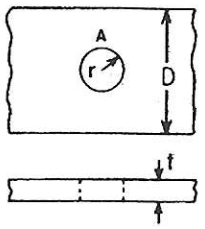
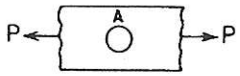
6. Circular hole in an infinite plate 	6a. Elastic stress, in-plane normal stress 	(a1) Uniaxial stress, $\sigma_2 = 0$ $\sigma_A = 3\sigma_1 \qquad \sigma_B = -\sigma_1$ (a2) Biaxial stress, $\sigma_2 = \sigma_1$ $\sigma_A = \sigma_B = 2\sigma_1$ (a3) Biaxial stress, $\sigma_2 = -\sigma_1$ (pure shear) $\sigma_A = -\sigma_B = 4\sigma_1$
	6b. Elastic stress, out-of-plane bending  Note: M_1 and M_2 are unit moments. See Table 11.2.	(b1) Simple bending, $M_2 = 0$ $\sigma_A = K_t \frac{6M_1}{t^2}$ where $K_t = 1.79 + \frac{0.25}{0.39 + (2r/t)} + \frac{0.81}{1 + (2r/t)^2} - \frac{0.26}{1 + (2r/t)^3}$ (b2) Cylindrical bending, $M_2 = \nu M_1$ $\sigma_A = K_t \frac{6M_1}{t^2}$ where $K_t = 1.85 + \frac{0.509}{0.70 + (2r/t)} - \frac{0.214}{1 + (2r/t)^2} + \frac{0.335}{1 + (2r/t)^3}$ for $\nu = 0.3$ (b3) Isotropic bending, $M_2 = M_1$ $\sigma_A = K_t \frac{6M_1}{t^2}$ where $K_t = 2$ (independent of r/t) (Refs. 1 and 26–30)
7. Central circular hole in a member of rectangular cross section 	7a. Elastic stress, axial tension 	$\sigma_{max} = \sigma_A = K_t \sigma_{nom}$ where $\sigma_{nom} = \frac{P}{t(D - 2r)}$ $K_t = 3.00 - 3.13\left(\frac{2r}{D}\right) + 3.66\left(\frac{2r}{D}\right)^2 - 1.53\left(\frac{2r}{D}\right)^3$ (Refs. 5 and 25)